

Abstract

This paper presents three definitions of equivalent stress calculation for micropolar solids. After a short review of the literature, the kinematical, equilibrium and constitutive equations are summarized. Applying the definition of the elastic strain energy, the paper gives three methods for the determination of equivalent stresses. After that, these are compared by a known cylinder torsion analytical example from literature.

Keywords

cosserat · micropolar · equivalent stress

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1 Introduction

Conventional formulation of continuum mechanics approaches cannot incorporate any intrinsic material length scales. However, real materials often exhibit a number of important length scales, which must be included in any realistic model: foams, granular materials, ceramics, biological tissues, fibers, particles, cellular solids and composites. Classical continuum mechanics considers the interaction of microstructural units of the material is described through stresses and displacements of material points. However interaction of grains may include rotation and associated couples-stress as well. Therefore the averaging scheme of classical continuum mechanics should consider rotation and couple stresses as well. This was the first Cosserat proposed by Eugene and Francois in their landmark publication. Their proposed theory was later reworked by Gunther and Mindlin who laid down the kinematics and statics of Cosserat continuum in a useful form for applied mechanics applications. Eringen coined the term micropolar continuum as a synonym for the Cosserat medium. Despite the obvious application to granular media, the Cosserat theory was first applied in soil mechanics applications when Mühlhaus and Vardoulakis applied the theory to localization. These authors observed that the characteristic dimension (thickness) of shear bands that are formed (in granular materials) in post localization regime is governed by the grain size. The definition of the equivalent stress is very important for the calculation of elasto-plastic theory, because the yield condition was usually based on them. In the published papers, De Borst [2], Lippmann [8], Besdo [1], Neuber [11] studied two dimensional cases. These formulas can not generalized for three dimensional. In tree dimension the equations contain more material parameters then in two dimennsional case. Forest [5] defined a general formula, but the constants what were used in the equations, are not identified by material parameters. This paper presents three different models for definitions of equivalent stress, which contain the material parameters [7], so these can be using also in numerical examples. All three definitions are based on the relation of general and uniaxial stress state, but these neglect different parts of the elastic strain energy. The yield condition used in elastoplastic theory can be determined from these

three equivalent stress models. This paper is organized as follows. The notation for equations will be introduced at the end of this section. Section 2 describes the fundamental equations for elastic, isotropic, micropolar solids. Section 3 contains the determination of the strain energy function. In Section 4, the three equivalent stress model definitions are presented. In Section 5, a numerical example is shown for the equivalent models, using a known analytical example from literature. Regarding notation, tensors are denoted by bold-face characters, the order of which is indicated in the text. The tensor product is denoted by \otimes , and the following symbolic operations apply: $\mathbf{a} : \mathbf{b} = a_{ij}b_{ij}$, $(\mathbf{A} : \mathbf{B})_{ijkl} = A_{ijmn}B_{mnkl}$ and $(\mathbf{C} : \mathbf{a})_{ij} = C_{ijkl}a_{kl}$, with the summation over repeated indices. The superscripts T and -1 denote transpose and inverse, and the prefix tr refers to the trace. The symbol $\|\mathbf{a}\| = \sqrt{\mathbf{a}^T : \mathbf{a}}$ is used to denote a norm of nonsymmetric second order tensor \mathbf{a} . The second-order unit tensor (Kronecker delta) is denoted by δ or δ_{ij} , the permutation tensor ϵ or ϵ_{ijk} .

2 Fundamental equations of linear isotropic, micropolar elasticity

In this section the constitutive equations for linear micropolar elasticity are briefly reviewed. Here we use the following notations: \mathbf{u} is the displacement vector, ϕ is the microrotation vector, ϵ and γ are the strain measures, \mathbf{t} denote the stress tensor and \mathbf{m} represent the couple stress tensor.

2.1 Equilibrium equations

Eringen [4] has published a comprehensive recapitulation of micropolar elasticity theory based largely on earlier papers written by him and his co-workers. His treatise will serve as the main source of the equations in this work. Only static problems will be considered. In the rectangular Cartesian tensor form, the equilibrium equations for stress t_{kl} and couple stress m_{kl} are written as:

$$t_{kl,k} + f_l = 0, \quad m_{kl,k} + \epsilon_{lmn}t_{mn} + l_l = 0. \quad (1)$$

where f_l and l_l denote the body force and the body couple.

2.2 Kinematic equations

At any material point of the continuum, we consider both a displacement and a micro rotation vector denoted by \mathbf{u} and ϕ , respectively. The associated Cosserat deformation ϵ and torsion-curvature tensor γ are written by [3] p. 104., eqn (5.1.7) as:

$$\epsilon = \text{grad}^T \mathbf{u} + (\epsilon \cdot \phi)^T, \quad (\epsilon_{ab} = u_{b,a} + \epsilon_{bac}\phi_c), \quad (2)$$

$$\gamma = \text{grad} \phi, \quad (\gamma_{ab} = \phi_{a,b}). \quad (3)$$

Note that ϵ and γ are non-symmetric tensors.

2.3 Constitutive relations

The constitutive equations for linear isotropic micropolar solid and their inverse are given by

$$\mathbf{t} = \lambda \text{tr} \epsilon \delta + (\mu + \kappa) \epsilon + \mu \epsilon^T, \quad (4)$$

$$\mathbf{m} = \alpha \text{tr} \gamma \delta + \beta \gamma + \gamma \gamma^T, \quad (5)$$

$$\epsilon = -\frac{\lambda}{(2\mu + \kappa)(3\lambda + 2\mu + \kappa)} \text{tr} \epsilon \delta + \frac{\mu + \kappa}{\kappa(2\mu + \kappa)} \mathbf{t} - \frac{\mu}{\kappa(2\mu + \kappa)} \mathbf{t}^T, \quad (6)$$

$$\gamma = -\frac{\alpha}{(\beta + \gamma)(3\alpha + \beta + \gamma)} \text{tr} \gamma \delta + \frac{\beta}{\beta^2 - \gamma^2} \mathbf{m} - \frac{\gamma}{\beta^2 - \gamma^2} \mathbf{m}^T, \quad (7)$$

where $\alpha, \beta, \gamma, \mu, \kappa, \lambda$ are material parameters. These equations can be re-written in the following form:

$$\mathbf{t} = \mathbf{A} : \epsilon, \quad (t_{ab} = A_{abcd}\epsilon_{cd}), \quad (8)$$

$$\mathbf{m}^T = \mathbf{B} : \gamma, \quad (m_{ab} = B_{bacd}\gamma_{cd}), \quad (9)$$

$$\epsilon = \mathbf{A}^{-1} : \mathbf{t}, \quad (\epsilon_{ab} = A_{abcd}^{-1}t_{cd}), \quad (10)$$

$$\gamma^T = \mathbf{B}^{-1} : \mathbf{m}, \quad (\gamma_{ab} = B_{bacd}^{-1}m_{cd}), \quad (11)$$

where the fourth order constitutive tensors A, B and their inverses are defined by

$$A_{abcd} = \lambda \delta_{ab}\delta_{cd} + (\mu + \kappa) \delta_{ac}\delta_{bd} + \mu \delta_{ad}\delta_{bc}, \quad (12)$$

$$B_{abcd} = \alpha \delta_{ab}\delta_{cd} + \beta \delta_{ad}\delta_{bc} + \gamma \delta_{ac}\delta_{bd}, \quad (13)$$

$$A_{abcd}^{-1} = -\frac{\lambda}{(2\mu + \kappa)(3\lambda + 2\mu + \kappa)} \delta_{ab}\delta_{cd} + \frac{\mu + \kappa}{\kappa(2\mu + \kappa)} \delta_{ac}\delta_{bd} - \frac{\mu}{\kappa(2\mu + \kappa)} \delta_{ad}\delta_{bc}, \quad (14)$$

$$B_{abcd}^{-1} = -\frac{\alpha}{(\beta + \gamma)(3\alpha + \beta + \gamma)} \delta_{ab}\delta_{cd} + \frac{\beta}{\beta^2 - \gamma^2} \delta_{ad}\delta_{bc} - \frac{\gamma}{\beta^2 - \gamma^2} \delta_{ac}\delta_{bd}. \quad (15)$$

2.4 Material parameters

Lakes [7] applying some experiments, defines six alternative constants from Eringen's moduli. These elastic parameters can be easily calculated from $\lambda, \mu, \kappa, \alpha, \beta, \gamma$ and vice versa by

$$\begin{aligned}
G_m &= \frac{1}{2}(2\mu + \kappa), \quad \nu_m = \frac{\lambda}{2\lambda + 2\mu + \kappa}, \\
\mathcal{N} &= \sqrt{\frac{\kappa}{2(\mu + \kappa)}}, \quad l_t = \sqrt{\frac{\beta + \gamma}{2\mu + \kappa}}, \\
l_b &= \sqrt{\frac{\gamma}{2(2\mu + \kappa)}}, \quad \Psi = \frac{\beta + \gamma}{\alpha + \beta + \gamma}, \\
\alpha &= \frac{2G_m(1 - \Psi)l_t^2}{\Psi}, \quad \beta = 2G_m(l_t^2 - 2l_b^2), \\
\gamma &= 4G_ml_b^2, \quad \lambda = \frac{2G_m\nu_m}{1 - 2\nu_m}, \\
\mu &= \frac{G_m(1 - 2\mathcal{N}^2)}{1 - \mathcal{N}^2}, \quad \kappa = \frac{2G_m\mathcal{N}^2}{1 - \mathcal{N}^2}.
\end{aligned} \tag{16}$$

Here G_m [force/length²] is the Shear modulus, ν_m [dimensionless] is the Poisson ratio, l_t and l_b [length] are the torsion and bending Characteristic length, \mathcal{N} [dimensionless] is the Coupling number and Ψ [dimensionless] is the Polar ratio. The effect of these parameters are studied by Nakamura [10].

3 Strain energy density function

The elastic strain energy function can be defined by the product of the stress and strain and product of couple-stress and curvature-strain as:

$$\begin{aligned}
U &= \frac{1}{2}\mathbf{t} : \boldsymbol{\varepsilon} + \frac{1}{2}\mathbf{m} : \boldsymbol{\gamma}^T = \frac{1}{2}\mathbf{t} : \mathbf{A}^{-1} : \\
&\mathbf{t} + \frac{1}{2}\mathbf{m} : \mathbf{B}^{-1} : \mathbf{m} = U_t(\mathbf{t}) + U_m(\mathbf{m}),
\end{aligned} \tag{17}$$

where the components $U_t(\mathbf{t})$ and $U_m(\mathbf{m})$ using the Eringen's elastic material parameters, are given by the following expressions:

$$\begin{aligned}
U_t(\mathbf{t}) &= \frac{1}{2} \left(-\frac{\lambda}{(2\mu + \kappa)(3\lambda + 2\mu + \kappa)} (\text{trt})^2 + \right. \\
&\left. \frac{\mu + \kappa}{\kappa(2\mu + \kappa)} \mathbf{t} : \mathbf{t} - \frac{\mu}{\kappa(2\mu + \kappa)} \mathbf{t} : \mathbf{t}^T \right),
\end{aligned} \tag{18}$$

$$\begin{aligned}
U_m(\mathbf{m}) &= \frac{1}{2} \left(-\frac{\alpha}{(\beta + \gamma)(3\alpha + \beta + \gamma)} (\text{trm})^2 + \right. \\
&\left. \frac{\beta}{\beta^2 - \gamma^2} \mathbf{m} : \mathbf{m}^T - \frac{\gamma}{\beta^2 - \gamma^2} \mathbf{m} : \mathbf{m} \right).
\end{aligned} \tag{19}$$

Dividing the stress and couple-stress into deviatoric and hydrostatic parts

$$\begin{aligned}
\mathbf{t} &= \mathbf{s} + \frac{1}{3}(\text{trt})\boldsymbol{\delta} \equiv \mathbf{s} + t_h\boldsymbol{\delta}, \\
\mathbf{m} &= \mathbf{m}_d + \frac{1}{3}(\text{trm})\boldsymbol{\delta} \equiv \mathbf{m}_d + m_h\boldsymbol{\delta},
\end{aligned} \tag{20}$$

then, the strain energy can be expressed in the following form:

$$U(\mathbf{t}, \mathbf{m}) = U_{td}(\mathbf{s}) + U_{md}(\mathbf{m}_d) + U_{th}(t_h) + U_{mh}(m_h), \tag{21}$$

where the deviatoric ($U_{td}(\mathbf{s})$ and $U_{md}(\mathbf{m}_d)$) and hydrostatic ($U_{th}(t_h)$ and $U_{mh}(m_h)$) parts of strain energy can be defined

as

$$U_{td}(\mathbf{s}) = \frac{1}{2} \left(\frac{\mu + \kappa}{\kappa(2\mu + \kappa)} \mathbf{s} : \mathbf{s} - \frac{\mu}{\kappa(2\mu + \kappa)} \mathbf{s} : \mathbf{s}^T \right), \tag{22}$$

$$U_{md}(\mathbf{m}_d) = \frac{1}{2} \left(\frac{\beta}{\beta^2 - \gamma^2} \mathbf{m}_d : \mathbf{m}_d^T - \frac{\gamma}{\beta^2 - \gamma^2} \mathbf{m}_d : \mathbf{m}_d \right), \tag{23}$$

$$U_{th}(t_h) = \frac{3}{2} \left(\frac{1}{\kappa + 3\lambda + 2\mu} t_h^2 \right), \tag{24}$$

$$U_{mh}(m_h) = \frac{3}{2} \left(\frac{1}{3\alpha + \beta + \gamma} m_h^2 \right). \tag{25}$$

Using the symmetric and anti-symmetric parts of the stress and couple-stress, the formulas (22) and (23) can be written as follows

$$U_{td}(\mathbf{s}) = \frac{1}{2} \left(\frac{1}{2\mu + \kappa} \mathbf{s}_S : \mathbf{s}_S + \frac{1}{\kappa} \mathbf{s}_A : \mathbf{s}_A \right), \tag{26}$$

$$U_{md}(\mathbf{m}_d) = \frac{1}{2} \left(\frac{1}{\beta + \gamma} \mathbf{m}_{dS} : \mathbf{m}_{dS} - \frac{1}{\beta - \gamma} \mathbf{m}_{dA} : \mathbf{m}_{dA} \right). \tag{27}$$

Using the material parameters defined in (2.4), Eringen [4] gives the necessary and sufficient conditions that the strain energy U be nonnegative as follows

$$\begin{aligned}
0 &\leq 3\lambda + 2\mu + \kappa, & 0 &\leq 2\mu + \kappa, & 0 &\leq \kappa, \\
0 &\leq 3\alpha + 2\beta + \gamma, & 0 &\leq \beta + \gamma, & 0 &\leq \gamma,
\end{aligned} \tag{28}$$

which can also be defined with the Lakes parameters as

$$\begin{aligned}
0 &\leq \Psi \leq \frac{3}{2}, \quad l_t \geq 0, \quad l_b \geq 0, \quad 2l_b \geq l_t, \\
0 &\leq \mathcal{N} \leq 1, \quad -1 \leq \nu_m \leq \frac{1}{2}.
\end{aligned} \tag{29}$$

4 The equivalent stress definitions

The formula (21) is the elastic strain energy function in a general, three dimensional case. For uniaxial stress state

$$\tilde{t}_{ab} = \sigma \delta_{1a} \delta_{1b}, \quad \tilde{s}_{ab} = \sigma \left(\delta_{1a} \delta_{1b} - \frac{1}{3} \delta_{ab} \right), \tag{30}$$

the strain energy can be expressed by

$$U^{1D} = U_{id}^{1D} + U_{ih}^{1D} = \frac{1}{3(2\mu + \kappa)}\sigma^2 + \frac{1}{6(\kappa + 3\lambda + 2\mu)}\sigma^2. \quad (31)$$

The equivalent stress definition are based on the comparison of uniaxial and the three dimensional cases.

Model 1.

This model supposes that the full strain energy is equivalent in uniaxial one, namely (31) and (21) are equals,

$$\left. \begin{aligned} U^{1D}(\sigma) &= U_{id}^{1D}(\sigma) + U_{ih}^{1D}(\sigma) \\ U(\mathbf{t}, \mathbf{m}) &= U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d) + U_{ih}(t_h) + U_{mh}(m_h) \end{aligned} \right\} U^{1D}(\sigma) = U(\mathbf{t}, \mathbf{m}). \quad (32)$$

From (32) it follows that

$$\sigma = \sqrt{\frac{6(\kappa + 3\lambda + 2\mu)(2\mu + \kappa)}{2(\kappa + 3\lambda + 2\mu) + \xi(2\mu + \kappa)} [U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d) + U_{ih}(t_h) + U_{mh}(m_h)]}. \quad (33)$$

Substitute Eqs. (22) – (25) and (31) into (33), the final form of the equivalent stress is

$$\bar{\sigma}^{(1)} = \sqrt{\frac{3}{2} a_1 [\mathbf{s}_S : \mathbf{s}_S + a_2 \mathbf{s}_A : \mathbf{s}_A + a_3 \mathbf{m}_{dS} : \mathbf{m}_{dS} + a_4 \mathbf{m}_{dA} : \mathbf{m}_{dA} + a_5 m_h^2 + a_6 t_h^2]}, \quad (34)$$

where the constants are defined by

$$a_1 = \frac{2(\kappa + 3\lambda + 2\mu)}{2(\kappa + 3\lambda + 2\mu) + (2\mu + \kappa)} \equiv \frac{2(1 + \nu_m)}{3}, \quad a_2 = \frac{2\mu + \kappa}{\kappa} \equiv \frac{1 - \mathcal{N}^2}{\mathcal{N}^2}, \quad (35)$$

$$a_3 = \frac{2\mu + \kappa}{\beta + \gamma} \equiv \frac{1}{l_t^2}, \quad a_4 = -\frac{2\mu + \kappa}{\beta - \gamma} \equiv \frac{1}{4l_b^2 - l_t^2}, \quad (36)$$

$$a_5 = \frac{3(2\mu + \kappa)}{3\alpha + \beta + \gamma} \equiv \frac{3\Psi}{(3 - 2\Psi)l_t^2}, \quad a_6 = \frac{3(2\mu + \kappa)}{\kappa + 3\lambda + 2\mu} \equiv \frac{9}{1 + \nu_m} - 6. \quad (37)$$

Model 2.

This model supposes that some parts of strain energy is not important for equivalent stress. The hydrostatical part from uniaxial case and the part of the general case related to stress, are not included. This means

$$\left. \begin{aligned} U^{1D}(\sigma) &= U_{id}^{1D}(\sigma) \\ U(\mathbf{s}, \mathbf{m}) &= U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d) + U_{mh}(m_h) \end{aligned} \right\} U^{1D}(\sigma) = U(\mathbf{s}, \mathbf{m}). \quad (38)$$

From (38) it follows that

$$\sigma = \sqrt{3(2\mu + \kappa) [U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d) + U_{mh}(m_h)]}. \quad (39)$$

Substituting Eqs. (22), (23), (25) and (31) into (38), the final form for equivalent stress is

$$\bar{\sigma}^{(2)} = \sqrt{\frac{3}{2} (\mathbf{s}_S : \mathbf{s}_S + a_2 \mathbf{s}_A : \mathbf{s}_A + a_3 \mathbf{m}_{dS} : \mathbf{m}_{dS} + a_4 \mathbf{m}_{dA} : \mathbf{m}_{dA} + a_5 m_h^2)}, \quad (40)$$

where constant a_2 , a_3 , a_4 and a_5 are the same as in the first model (35) – (37).

Model 3.

This model contains more suppositions than the second one. It neglects the hydrostatical part from uniaxial case again and hydrostatical part of stress and couple stress part of general case. This means that (31) without U_{th}^{1D} is equal to (21) without U_{th} and U_{mh} .

$$\left. \begin{aligned} U^{1D}(\sigma) &= U_{id}^{1D}(\sigma) \\ U(\mathbf{s}, \mathbf{m}_d) &= U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d) \end{aligned} \right\} U^{1D}(\sigma) = U(\mathbf{s}, \mathbf{m}_d). \quad (41)$$

From (41) it follows that

$$\sigma = \sqrt{3(2\mu + \kappa)(U_{id}(\mathbf{s}) + U_{md}(\mathbf{m}_d))}. \quad (42)$$

Combining Eqs (22), (23) and (31) into (42), the finally form for equivalent stress is

$$\bar{\sigma}^{(3)} = \sqrt{\frac{3}{2}(\mathbf{s}_S : \mathbf{s}_S + a_2 \mathbf{s}_A : \mathbf{s}_A + a_3 \mathbf{m}_{dS} : \mathbf{m}_{dS} + a_4 \mathbf{m}_{dA} : \mathbf{m}_{dA})}, \quad (43)$$

where constants a_2 , a_3 and a_4 are defined by (35), (36). Note that Forest [5] presented a similar formula for equivalent stress, but he has applied the total couple-stress instead of its deviatoric part.

5 Numerical example. Torsion and tension of solid circular cylinder

The boundary-value problems deal with cylinders of radius R and length c subjected to either axial tension P and torsion T . The analytical solution in cylindrical coordinates (r, Θ, z) is given by Gauthier and Jashman [6]. The boundary conditions are: $r = R$, $P = \int_A t_{zz} dA$, $T = \int_A (rt_{z\Theta} + m_{zz}) dA$ and $z = 0$. The solution is given by

$$t_{zz} = \frac{P}{A}, \quad (44)$$

$$t_{\theta z} = (2\mu + \kappa) \frac{rC_1}{2} - \kappa C_9 I_1(pr), \quad (45)$$

$$t_{z\theta} = (2\mu + \kappa) \frac{rC_1}{2} + \kappa C_9 I_1(pr), \quad (46)$$

$$m_{rr} = (\alpha + \beta + \gamma) p C_9 I_0(pr) - \frac{\beta + \gamma}{r} \left(C_9 I_1(pr) + \frac{rC_1}{2} \right), \quad (47)$$

$$m_{\theta\theta} = \alpha p C_9 I_0(pr) + \frac{\beta + \gamma}{r} \left(C_9 I_1(pr) - \frac{rC_1}{2} \right), \quad (48)$$

$$m_{zz} = \alpha p C_9 I_0(pr) + (\beta + \gamma) C_1, \quad (49)$$

$$C_1 = 2C_9 \left(\frac{\alpha + \beta + \gamma}{\beta + \gamma} p I_0(pR) - \frac{1}{R} I_1(pR) \right), \quad (50)$$

$$C_9 = \frac{T}{2A \left((\alpha + \beta + \gamma) \left(\frac{3}{2} + k \right) p I_0(pR) - (2 + k)(\beta + \gamma) \frac{I_1(pR)}{R} \right)}, \quad (51)$$

$$p^2 = \frac{2\kappa}{\alpha + \beta + \gamma}, \quad k = \frac{R^2(\kappa + 2\mu)}{4(\beta + \gamma)}, \quad (52)$$

where $I_n ()$ is the modified Bessel function of the first kind of order n.

For this problem, using (44) – (52) and (34), (40), (43) the equivalent stresses can be expressed by

$$\bar{\sigma}^{(1)} = \left\{ \frac{2(1 + \nu_m)}{3} \left(\frac{(m_{rr} - m_{\theta\theta})^2 + (m_{rr} - m_{zz})^2 + (m_{zz} - m_{\theta\theta})^2}{2l_t^2} + \frac{3(t_{z\theta} - t_{\theta z})^2}{4N^2} + 3t_{z\theta}t_{\theta z} + \frac{\Psi(m_{rr} + m_{\theta\theta} + m_{zz})^2}{2(2 - \Psi)l_t^2} + \frac{3t_{zz}^2}{2(1 + \nu_m)} \right) \right\}^{\frac{1}{2}}, \quad (53)$$

$$\bar{\sigma}^{(2)} = \left\{ \frac{(m_{rr} - m_{\theta\theta})^2 + (m_{rr} - m_{zz})^2 + (m_{zz} - m_{\theta\theta})^2}{2l_t^2} + \frac{\Psi(m_{rr} + m_{\theta\theta} + m_{zz})^2}{2(2 - \Psi)l_t^2} + \frac{3(t_{z\theta} - t_{\theta z})^2}{4N^2} + 3t_{z\theta}t_{\theta z} + t_{zz}^2 \right\}^{\frac{1}{2}}, \quad (54)$$

$$\bar{\sigma}^{(3)} = \left\{ \frac{(m_{rr} - m_{\theta\theta})^2 + (m_{rr} - m_{zz})^2 + (m_{zz} - m_{\theta\theta})^2}{2l_t^2} + \frac{3(t_{z\theta} - t_{\theta z})^2}{4N^2} + 3t_{z\theta}t_{\theta z} + t_{zz}^2 \right\}^{\frac{1}{2}}, \quad (55)$$

where $\bar{\sigma}^1$, $\bar{\sigma}^2$ and $\bar{\sigma}^3$ correspond to the models 1-3, respectively. Using the material parameters $G_m = 1033 [MPa]$, $\nu_m = 0.34$, $l_t = 0.65 [mm]$, $l_b = 0.4 [mm]$, $N = 0.6$, $\Psi = 1.5$, the distribution of equivalent stress on radius are presented by the following figure.

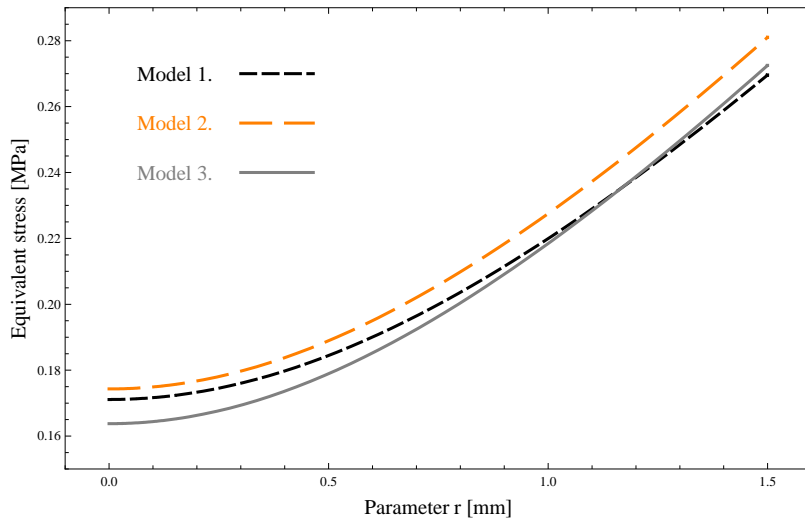


Fig. 1. Variation of equivalent stresses vs. radius of the cylinder according to Models 1-3

Fig. 1 shows that for the torsion problem, there are no significant differences between the equivalent stress models which are introduced in this paper.

6 Conclusion

A brief summary of the fundamental equations of elastic, micropolar solids is given in this paper. It defines the elastic strain energy which is based on the additive decomposition of stress and couple stress parts. Using this definition, the paper introduces three different models for equivalent stress, which are based on the comparison of uniaxial and general stress state. Furthermore numerical differences have been shown between the three models applying a known analytical solution (torsion of a circular cylinder). The comparison shows that there are no significant differences with the used material parameters for this problem. This formulas can be applicable to yield condition for the elastoplastic micropolar model.

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